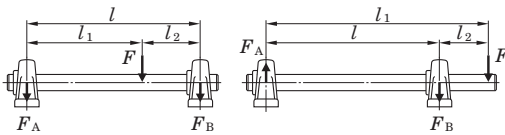


5 Bearing Load

5.2 Distribution of Bearing Load

In order to determine the radial load distribution to each bearing attached to a shaft, use the procedure shown below. Use the load factors shown in **Table 5.1** to account for vibration and impact.

A standard radial ball bearing bears an axial load component in addition to the radial component. The total vectored load can be calculated by taking the square root of the sum of the squares of each load as shown in the previous calculation.



$$F_A = \frac{l_2}{l} \cdot F \dots\dots\dots (5.7)$$

$$F_B = \frac{l_1}{l} \cdot F \dots\dots\dots (5.8)$$

Fig. 5.1 Distribution of load to bearings

5.3 Dynamic Equivalent Load

In many cases, a bearing is exposed to the combined vector load of both radial and axial load components. It may also be used under more severe conditions such as vibration and shock load. In this case, a direct comparison to the dynamic load rating is not appropriate.

In such a case, find the load equivalent to a direct radial load only and compare this with the basic dynamic load rating.

The converted virtual load is called dynamic equivalent load (*P*).

5.3.1 Calculation of dynamic equivalent load

The dynamic equivalent radial load (*P_r*) of a bearing that bears radial and axial loads as well as vibration and impact is found by the following formula.

$$P_r = XF_r + YF_a \dots\dots\dots (5.9)$$

Whereas,

P_r: Dynamic equivalent radial load, N

F_r: Radial load, N

F_a: Axial load, N

X: Radial load factor (see **Table 5.4**)

Y: Axial load factor (see **Table 5.4**)

Table 5.4 Radial load factor (X) and axial load factor (Y)

| $\frac{f_0 F_a}{C_{0r}}$ | <i>e</i> | $F_a / F_r \leq e$ | | $F_a / F_r > e$ | |
|--------------------------|----------|--------------------|----------|-----------------|----------|
| | | <i>X</i> | <i>Y</i> | <i>X</i> | <i>Y</i> |
| 0.172 | 0.19 | 1 | 0 | 0.56 | 2.30 |
| 0.345 | 0.22 | | | | 1.99 |
| 0.689 | 0.26 | | | | 1.71 |
| 1.03 | 0.28 | | | | 1.55 |
| 1.38 | 0.30 | | | | 1.45 |
| 2.07 | 0.34 | | | | 1.31 |
| 3.45 | 0.38 | | | | 1.15 |
| 5.17 | 0.42 | | | | 1.04 |
| 6.89 | 0.44 | | | | 1.00 |

Remarks 1. *C_{0r}* (basic static radial load rating) and *f₀* (factor) are shown in the dimensional tables.
2. If *f₀ F_a / C_{0r}* does not conform to the table above, find by interpolation.

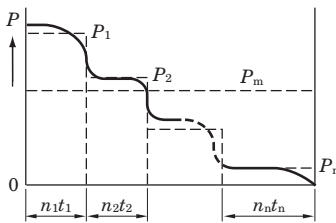
5.3.2 Average dynamic equivalent load in the case of fluctuating loads

If the level or direction of the load applied to a bearing is fluctuating, it is necessary to find the average dynamic equivalent load to calculate the bearing life.

Table 5.5 shows the method of finding the average dynamic equivalent load under various types of fluctuating conditions.

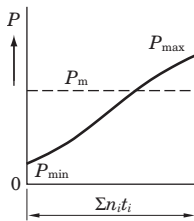
Table 5.5 Calculation of average dynamic equivalent load in case of fluctuated load

(1) Graduated fluctuation



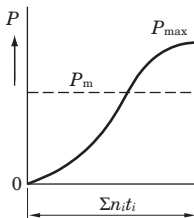
$$P_m = \sqrt[p]{\frac{P_1^p n_1 t_1 + P_2^p n_2 t_2 + \dots + P_n^p n_n t_n}{n_1 t_1 + n_2 t_2 + \dots + n_n t_n}} \quad (5.10)$$

(2) Monotone fluctuation



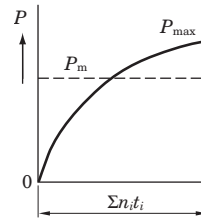
$$P_m = \frac{P_{min} + 2 P_{max}}{3} \quad (5.11)$$

(3) Sine curve fluctuation



$$P_m = 0.68 P_{max} \quad (5.12)$$

(4) Sine curve fluctuation (upper half of sine curve)



$$P_m = 0.75 P_{max} \quad (5.13)$$

Whereas,

- P_m : Average dynamic equivalent load, N
- P_1 : Dynamic equivalent load actuating for t_1 hours at rotating speed of n_1 , N
- P_2 : Dynamic equivalent load actuating for t_2 hours at rotating speed of n_2 , N
- ...
- P_n : Dynamic equivalent load actuating for t_n hours at rotating speed of n_n , N
- P_{min} : Minimum dynamic equivalent load, N
- P_{max} : Maximum dynamic equivalent load, N
- $\Sigma n_i t_i$: Total rotating frequency for t_1 to t_i hours

5.4 Basic Static Load Rating and Static Equivalent Load

5.4.1 Basic static load rating

If a bearing is exposed to excessive static or impact load even when running at low rotational speed, partial permanent deformation occurs to the contact surface of the raceways of the bearing. The amount of permanent deformation increases with increased loads, and at some point, the bearing will no longer rotate smoothly.

The basic static load rating of a bearing is the static load that generates the calculated contact stresses shown below at the center of the contact surfaces of the raceways.

- (1) Self aligning ball bearings 4,600 MPa
- (2) Other ball bearings
(mounted ball bearings included) 4,200 MPa
- (3) Roller bearings 4,000 MPa

The total permanent deformation that occurs to the raceways and the balls under the above critical contact stresses is 0.0001 times the diameter of the ball.

In ball bearing units, this is indicated as the basic static radial load rating (C_{0r}) and these values are shown in the dimensional tables.

5 Bearing Load

5.4.2 Static equivalent loads

Static equivalent load is the equivalent of the combined (vectored) load converted to the equivalent direct radial load. The term "static" refers to no rotation or very little rotation.

Static equivalent radial load (P_{0r}) can be calculated by using the formula below.

$$P_{0r} = 0.6 F_r + 0.5 F_a \quad (5.14)$$

$$P_{0r} = F_r \quad (5.15)$$

Whereas,

P_{0r} : Static equivalent radial load, N

F_r : Radial load, N

F_a : Axial load, N

5.4.3 Safety factor

The static equivalent load that can be withstood by a bearing, in addition to the above considerations, is sometimes dependent upon unforeseen conditions in the operating environment. Therefore, a safety factor is always built in to insure success in the application.

$$f_s = \frac{C_{0r}}{P_{0r}} \quad (5.16)$$

Whereas,

f_s : Safety factor (see **Table 5.6**)

C_{0r} : Basic static radial load rating, N

P_{0r} : Static equivalent radial load, N

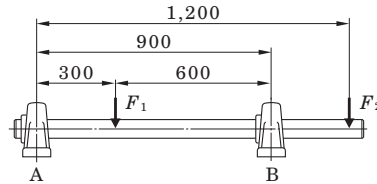
Table 5.6 Safety factor f_s (recommended)

| Operating conditions | | f_s (Min.) |
|--|------------------------------------|--------------|
| Being rotated | High rotating accuracy is required | 2 |
| | Ordinary operating conditions | 1 |
| | Impact | 1.5 |
| Not always being rotated (sometimes oscillated) | Ordinary operating conditions | 0.5 |
| | Impact, unevenly distributed load | 1 |

5.5 Example of Applied Calculation

Example 1 Distributing load

Find the load applied to the bearing A and bearing B, if the radial load F_1 ($F_1 = 1.5$ kN) and F_2 ($F_2 = 4.5$ kN) are applied.



- (1) Find the radial load F_{1A} applied to the bearing A by F_1 , with **Formula (5.7)** and **Formula (5.8)**.

$$F_{1A} = \frac{600}{900} \times 1.5 = 1.0 \text{ (kN)}$$

In a similar manner, find the radial load F_{2A} applied to the bearing A by F_2 .

$$F_{2A} = - \frac{1,200 - 900}{900} \times 4.5 = -1.5 \text{ (kN)}$$

Remark: Negative load is the upward load.

Radial load F_A applied to the bearing A:

$$F_A = F_{1A} + F_{2A} = 1.0 + (-1.5) = -0.5 \text{ (kN)}$$

- (2) In a similar manner to (1), find the radial load F_B applied to the bearing B.

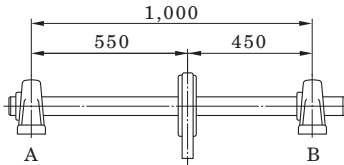
$$F_{1B} = \frac{300}{900} \times 1.5 = 0.5 \text{ (kN)}$$

$$F_{2B} = \frac{1,200}{900} \times 4.5 = 6.0 \text{ (kN)}$$

$$F_B = F_{1B} + F_{2B} = 0.5 + 6.0 = 6.5 \text{ (kN)}$$

Example 2 Calculating load by V-belt transmission

Find the load applied to the bearing A and bearing B when the shaft is driven by the V-belt, transmission power W is 7.5 kW ($W = 7.5$ kW), rotating speed n is 300 min^{-1} ($n = 300$ min^{-1}), effective diameter of pulley D_p is 300 mm ($D_p = 300$ mm).



- (1) Find the load actually applied to the pulley shaft F_b with **Formula (5.2)**.

From **Table 5.1**, load factor f_w is 1.2 ($f_w = 1.2$), and the belt factor f_b is 2.5 ($f_b = 2.5$), from **Table 5.2**.

$$F_b = \frac{19.1 \times 10^6 W}{D_p \cdot n} \cdot f_w \cdot f_b$$

$$= \frac{19.1 \times 10^6 \times 7.5}{300 \times 300} \times 1.2 \times 2.5 = 4.78 \text{ (kN)}$$

- (2) Find the load actually applied to the bearing A and bearing B (F_A and F_B) with **Formulas (5.7) and (5.8)**.

$$F_A = \frac{450}{1,000} \times 4.78 = 2.15 \text{ (kN)}$$

$$F_B = \frac{550}{1,000} \times 4.78 = 2.63 \text{ (kN)}$$

Example 3 Calculating dynamic equivalent radial load

Find the dynamic equivalent radial load P_r when the radial load F_r , 1.5 kN ($F_r = 1.5$ kN), and the axial load F_a , 0.85 kN, ($F_a = 0.85$ kN) are applied to the pillow type unit UCP306J (bearing UC306).

- (1) Find the radial load factor (X) and the axial load factor (Y) with using the static radial load rating C_{0r} of UCP306J (bearing UC306), 15.0 kN ($C_{0r} = 15.0$ kN), and **Table 5.4**.

Find the solutions of the following formulas:

$$\frac{f_0 F_a}{C_{0r}} = \frac{13.3 \times 0.85}{15.0} = 0.754, e = 0.264$$

$$\frac{F_a}{F_r} = \frac{0.85}{1.5} = 0.567 > e (0.264)$$

Therefore, $X = 0.56$, $Y = 1.68$

- (2) Find the dynamic equivalent radial load P_r with **Formula (5.9)**.

$$P_r = XF_r + YF_a = 0.56 \times 1.5 + 1.68 \times 0.85$$

$$= 2.27 \text{ (kN)}$$

Example 4 Calculating bearing life

Under the conditions shown in **Example 3**, find the bearing life L_{10h} when a bearing is used for a blower with a rotating speed n , 1,000 min^{-1} .

- (1) Select the load factor f_w is 1.2 ($f_w = 1.2$) from **Table 5.1**, and find the bearing load P_r .

$$P_r = f_w \cdot F = 1.2 \times 2.27 = 2.72 \text{ (kN)}$$

- (2) The dynamic radial load rating of UCP306J (bearing UC306), C_r , is 26.7 kN ($C_r = 26.7$ kN), and calculate the bearing life L_{10h} with the **Formula (4.2)**.

$$L_{10h} = \frac{10^6}{60n} \cdot \left(\frac{C_r}{P_r}\right)^3 = \frac{10^6}{60 \times 1,000} \times \left(\frac{26.7}{2.72}\right)^3$$

$$\approx 15,800 \text{ (hr)}$$

- (3) Calculate bearing life L_{10h} with the nomogram shown in **Fig. 4.1**.

When the rotating speed n is 1,000 min^{-1} ($n = 1,000$ min^{-1}), rotating factor f_n is 0.32 ($f_n = 0.32$). Next, find the life factor f_h by speed factor f_n , dynamic radial load rating of bearing C_r , and the bearing load P_r .

$$\text{Life factor } f_h = f_n \cdot \frac{C_r}{P_r} = 0.32 \times \frac{26.7}{2.72} = 3.14$$

From life factor f_h , bearing life $L_{10h} \approx 16,000$ hours.

Example 5 Selecting ball bearing units

If a bearing is operated under the following conditions, select the flange type unit (UCF) with at least two years (5,000 hours) or longer rating life: rotating speed of shaft n is 1,500 min^{-1} ($n = 1,500$ min^{-1}), and radial load F_r is 5 kN ($F_r = 5$ kN). The radial load F_r includes the load factor and gear factor.

- (1) From the nomogram shown in **Fig. 4.1**, when life time L_h is 5,000 hr ($L_h = 5,000$ hr), life factor f_h can be found as 2.16 ($f_h \approx 2.16$), and speed factor f_n can be found as 0.28 ($f_n \approx 0.28$) when the rotating speed n is 1,500 min^{-1} ($n = 1,500$ min^{-1}).

$$\text{Dynamic radial load rating } C_r = F_r \cdot \frac{f_h}{f_n} = 5 \times \frac{2.16}{0.28}$$

$$\approx 38.6 \text{ (kN)}$$

- (2) Find the flange type unit that meets the following condition: dynamic radial load rating C_r is 38.6 kN ($C_r = 38.6$ kN). For the 200 series, UCF211J (dynamic radial load rating C_r is 43.4 kN ($C_r = 43.4$ kN)) can be selected.



5 Bearing Load

Example 6 Selecting pillow type units for low speed

If a bearing is used for a dolly under the following conditions, select the pillow type unit (UCP) with 10,000 hours rating life: radial load F_r is 12 kN ($F_r = 12$ kN), and rotating speed is 8 min^{-1} .

- (1) Find the required dynamic radial load rating C_r with using **Formulas (4.4)** and **(4.5)**.

$$\text{Speed factor } f_n = (0.03n)^{-1/p} = (0.03 \times 8)^{-1/3} \approx 1.61$$

$$\text{Life factor } f_h = \left(\frac{L_{10h}}{500}\right)^{1/p} = \left(\frac{10,000}{500}\right)^{1/3} \approx 2.71$$

$$\text{Dynamic radial load rating } C_r = P_r \cdot \frac{f_h}{f_n} = 12 \times \frac{2.71}{1.61} \approx 20.2 \text{ (kN)}$$

- (2) From **Table 5.6**, define safe factor f_s as 2 ($f_s = 2$), and find the static radial load rating of bearing required C_{0r} .

$$C_{0r} = f_s \cdot P_r = 2 \times 12 = 24 \text{ (kN)}$$

- (3) The unit is used for a dolly, and vibration or impact may occur. Thus, select UCP308J ($C_r = 40.7$ kN, $C_{0r} = 24.0$ kN).

Example 7 Calculating bearing life in high temperature applications

Find the bearing life if the heat resistant pillow type unit (UCP215D1K2) is operated under the following conditions: operating temperature is 175°C , radial load F_r is 4 kN ($F_r = 4$ kN), and the rotating speed n is 800 min^{-1} ($n = 800 \text{ min}^{-1}$). Note that the radial load F_r includes load factor and gear factor.

- (1) From **Table 4.1**, find the dynamic load rating C_r in the case that a bearing is used at 175°C .

$$C_r = 67.4 \times 0.95 = 64.0 \text{ (kN)}$$

Find the bearing life L_{10h} using **Formula (4.2)**.

$$L_{10h} = \frac{10^6}{60n} \cdot \left(\frac{C_r}{P_r}\right)^3 = \frac{10^6}{60 \times 800} \times \left(\frac{64.0}{4}\right)^3 \approx 85,000 \text{ (hr)}$$

- (2) If a bearing unit is operated at 175°C , grease is degraded faster, and it cannot be used without lubrication. Supply grease at intervals specified in **Table 14.4**.
- (3) If the shaft experiences axial expansion due to heat, install a fixed bearing unit on one end of the assembly and install floating bearing unit on the other side that allows the shaft to move freely through the bore of the bearing. More information is offered in **Section 9**. (see “9 Design of shaft and base”).

Example 8 Calculating grease life

Find the grease life for pillow type unit UCP204J (bearing UC204) under the following conditions: radial load F_r is 1 kN ($F_r = 1$ kN), and rotating speed n is 800 min^{-1} ($n = 800 \text{ min}^{-1}$). Note that the radial load F_r includes load factor and belt factor. Operating temperature of the bearing should be 40°C .

Find the grease life L using **Formula (4.7)**.

$$\begin{aligned} \log L &= 6.10 - 4.40 \times 10^{-6} d_m n - 2.50 \left(\frac{P_r}{C_r} - 0.05\right) \\ &\quad - (0.021 - 1.80 \times 10^{-8} d_m n) T \\ &= 6.10 - 4.40 \times 10^{-6} \times 12.5 \times 10^4 \\ &\quad - 2.50 \left(\frac{1}{12.8} - 0.05\right) \\ &\quad - (0.021 - 1.80 \times 10^{-8} \times 12.5 \times 10^4) \times 50 \\ &= 4.542 \\ L &\approx 34,800 \text{ (hr)} \end{aligned}$$

Example 9 Calculating life of bearing units in case of non-lubrication

Find the life of a bearing unit in the case that it is operated under the conditions shown in **Example 8**, but without lubrication.

- (1) Find the rating life of bearings L_{10h} using **Formula (4.2)**.

$$L_{10h} = \frac{10^6}{60n} \cdot \left(\frac{C_r}{P_r}\right)^3 = \frac{10^6}{60 \times 800} \times \left(\frac{12.8}{1}\right)^3 \approx 43,700 \text{ (hr)}$$

- (2) Compare the grease life L shown in **Example 8** to the rating life of bearings L_h . Then, grease life L is shorter than the bearing rating life. Therefore, life of a bearing unit should be the same as the grease life L , 34,800 hours ($L = 34,800$ hours).



Table 5.7 (1) Radial Load/Speed Chart

Normal Duty

Unit : lbf

| SHAFT SIZE | L ₁₀ hours | Allowable Radial Load at Various RPM | | | | | | | | | | | | | | | | | | | | | |
|---|-----------------------|--------------------------------------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|
| | | RPM (min ⁻¹) | | | | | | | | | | | | | | | | | | | | | |
| | | 50 | 100 | 150 | 300 | 500 | 750 | 1000 | 1500 | 2000 | 2500 | 3000 | 3500 | 4000 | 4500 | 5000 | 5500 | 6000 | 6500 | 7000 | 7500 | 8000 | 8500 |
| 1/2" 5/8" 3/4" 12 mm - 20 mm | 10000 | 927 | 736 | 643 | 510 | 430 | 376 | 341 | 298 | 271 | 252 | 237 | 225 | 215 | 207 | 200 | 193 | 188 | 183 | 179 | 174 | 171 | 167 |
| | 25000 | 683 | 542 | 474 | 376 | 317 | 277 | 252 | 220 | 200 | 185 | 174 | 166 | 158 | 152 | 147 | 143 | 138 | 135 | 132 | 129 | 126 | 123 |
| | 50000 | 542 | 430 | 376 | 298 | 252 | 220 | 200 | 174 | 158 | 147 | 138 | 132 | 126 | 121 | 117 | 113 | 110 | 107 | 104 | 102 | 100 | 98 |
| | 70000 | 485 | 385 | 336 | 267 | 225 | 196 | 179 | 156 | 142 | 132 | 124 | 118 | 112 | 108 | 104 | 101 | 98 | 96 | 93 | 91 | 89 | 87 |
| | 100000 | 430 | 341 | 298 | 237 | 200 | 174 | 158 | 138 | 126 | 117 | 110 | 104 | 100 | 96 | 93 | 90 | 87 | 85 | 83 | 81 | 79 | 78 |
| 7/8" 15/16" 1" 25 mm | 10000 | 1014 | 805 | 703 | 558 | 471 | 411 | 373 | 326 | 296 | 275 | 259 | 246 | 235 | 226 | 218 | 212 | 206 | 200 | 195 | 191 | | |
| | 25000 | 747 | 593 | 518 | 411 | 347 | 303 | 275 | 240 | 218 | 203 | 191 | 181 | 173 | 167 | 161 | 156 | 151 | 147 | 144 | 141 | | |
| | 50000 | 593 | 471 | 411 | 326 | 275 | 240 | 218 | 191 | 173 | 161 | 151 | 144 | 138 | 132 | 128 | 124 | 120 | 117 | 114 | 112 | | |
| | 70000 | 530 | 421 | 367 | 292 | 246 | 215 | 195 | 171 | 155 | 144 | 135 | 129 | 123 | 118 | 114 | 111 | 107 | 105 | 102 | 100 | | |
| | 100000 | 471 | 373 | 326 | 259 | 218 | 191 | 173 | 151 | 138 | 128 | 120 | 114 | 109 | 105 | 101 | 98 | 95 | 93 | 91 | 89 | | |
| 1-1/8" 1-3/16" 1-1/4" 30 mm | 10000 | 1412 | 1121 | 979 | 777 | 655 | 573 | 520 | 454 | 413 | 383 | 361 | 343 | 328 | 315 | 304 | 295 | 286 | | | | | |
| | 25000 | 1040 | 826 | 721 | 573 | 483 | 422 | 383 | 335 | 304 | 282 | 266 | 252 | 241 | 232 | 224 | 217 | 211 | | | | | |
| | 50000 | 826 | 655 | 573 | 454 | 383 | 335 | 304 | 266 | 241 | 224 | 211 | 200 | 192 | 184 | 178 | 172 | 167 | | | | | |
| | 70000 | 738 | 586 | 512 | 406 | 343 | 299 | 272 | 238 | 216 | 200 | 189 | 179 | 171 | 165 | 159 | 154 | 150 | | | | | |
| | 100000 | 655 | 520 | 454 | 361 | 304 | 266 | 241 | 211 | 192 | 178 | 167 | 159 | 152 | 146 | 141 | 137 | 133 | | | | | |
| 1-1/4" 1-5/16" 1-3/8" 1-7/16" 35 mm | 10000 | 1861 | 1477 | 1290 | 1024 | 864 | 755 | 686 | 599 | 544 | 505 | 475 | 452 | 432 | 415 | 401 | 388 | | | | | | |
| | 25000 | 1371 | 1088 | 951 | 755 | 636 | 556 | 505 | 441 | 401 | 372 | 350 | 333 | 318 | 306 | 295 | 286 | | | | | | |
| | 50000 | 1088 | 864 | 755 | 599 | 505 | 441 | 401 | 350 | 318 | 295 | 278 | 264 | 253 | 243 | 234 | 227 | | | | | | |
| | 70000 | 973 | 772 | 675 | 535 | 452 | 394 | 358 | 313 | 284 | 264 | 248 | 236 | 226 | 217 | 210 | 203 | | | | | | |
| | 100000 | 864 | 686 | 599 | 475 | 401 | 350 | 318 | 278 | 253 | 234 | 221 | 210 | 200 | 193 | 186 | 180 | | | | | | |
| 1-1/2" 1-9/16" 40 mm | 10000 | 2107 | 1672 | 1461 | 1160 | 978 | 854 | 776 | 678 | 616 | 572 | 538 | 511 | 489 | 470 | 454 | | | | | | | |
| | 25000 | 1553 | 1232 | 1077 | 854 | 721 | 630 | 572 | 500 | 454 | 421 | 397 | 377 | 360 | 346 | 334 | | | | | | | |
| | 50000 | 1232 | 978 | 854 | 678 | 572 | 500 | 454 | 397 | 360 | 334 | 315 | 299 | 286 | 275 | 265 | | | | | | | |
| | 70000 | 1102 | 874 | 764 | 606 | 511 | 447 | 406 | 355 | 322 | 299 | 281 | 267 | 256 | 246 | 237 | | | | | | | |
| | 100000 | 978 | 776 | 678 | 538 | 454 | 397 | 360 | 315 | 286 | 265 | 250 | 237 | 227 | 218 | 211 | | | | | | | |
| 1-5/8" 1-11/16" 1-3/4" 45 mm | 10000 | 2469 | 1960 | 1712 | 1359 | 1146 | 1001 | 910 | 795 | 722 | 670 | 631 | 599 | 573 | 551 | | | | | | | | |
| | 25000 | 1819 | 1444 | 1261 | 1001 | 844 | 738 | 670 | 586 | 532 | 494 | 465 | 441 | 422 | 406 | | | | | | | | |
| | 50000 | 1444 | 1146 | 1001 | 795 | 670 | 586 | 532 | 465 | 422 | 392 | 369 | 350 | 335 | 322 | | | | | | | | |
| | 70000 | 1291 | 1025 | 895 | 710 | 599 | 523 | 476 | 415 | 377 | 350 | 330 | 313 | 300 | 288 | | | | | | | | |
| | 100000 | 1146 | 910 | 795 | 631 | 532 | 465 | 422 | 369 | 335 | 311 | 293 | 278 | 266 | 256 | | | | | | | | |
| 1-7/8" 1-15/16" 2" 50 mm | 10000 | 2542 | 2017 | 1762 | 1399 | 1180 | 1031 | 936 | 818 | 743 | 690 | 649 | 617 | 590 | | | | | | | | | |
| | 25000 | 1873 | 1486 | 1298 | 1031 | 869 | 759 | 690 | 603 | 548 | 508 | 478 | 454 | 435 | | | | | | | | | |
| | 50000 | 1486 | 1180 | 1031 | 818 | 690 | 603 | 548 | 478 | 435 | 403 | 380 | 361 | 345 | | | | | | | | | |
| | 70000 | 1329 | 1055 | 921 | 731 | 617 | 539 | 489 | 428 | 389 | 361 | 339 | 322 | 308 | | | | | | | | | |
| | 100000 | 1180 | 936 | 818 | 649 | 548 | 478 | 435 | 380 | 345 | 320 | 301 | 286 | 274 | | | | | | | | | |
| 2" 2-1/8" 2-3/16" 55 mm | 10000 | 3143 | 2494 | 2179 | 1729 | 1459 | 1274 | 1158 | 1011 | 919 | 853 | 803 | 763 | | | | | | | | | | |
| | 25000 | 2316 | 1838 | 1606 | 1274 | 1075 | 939 | 853 | 745 | 677 | 629 | 591 | 562 | | | | | | | | | | |
| | 50000 | 1838 | 1459 | 1274 | 1011 | 853 | 745 | 677 | 591 | 537 | 499 | 469 | 446 | | | | | | | | | | |
| | 70000 | 1643 | 1304 | 1139 | 904 | 763 | 666 | 605 | 529 | 480 | 446 | 420 | 399 | | | | | | | | | | |
| | 100000 | 1459 | 1158 | 1011 | 803 | 677 | 591 | 537 | 469 | 427 | 396 | 373 | 354 | | | | | | | | | | |
| 2-1/4" 2-3/8" 2-7/16" 60 mm | 10000 | 3794 | 3012 | 2631 | 2088 | 1761 | 1539 | 1398 | 1221 | 1109 | 1030 | 969 | 921 | | | | | | | | | | |
| | 25000 | 2796 | 2219 | 1938 | 1539 | 1298 | 1134 | 1030 | 900 | 817 | 759 | 714 | 678 | | | | | | | | | | |
| | 50000 | 2219 | 1761 | 1539 | 1221 | 1030 | 900 | 817 | 714 | 649 | 602 | 567 | 538 | | | | | | | | | | |
| | 70000 | 1984 | 1574 | 1375 | 1092 | 921 | 804 | 731 | 638 | 580 | 538 | 507 | 481 | | | | | | | | | | |
| | 100000 | 1761 | 1398 | 1221 | 969 | 817 | 714 | 649 | 567 | 515 | 478 | 450 | 427 | | | | | | | | | | |
| 2-1/2" 65 mm | 10000 | 4142 | 3287 | 2872 | 2279 | 1923 | 1679 | 1526 | 1333 | 1211 | 1124 | 1058 | | | | | | | | | | | |
| | 25000 | 3052 | 2422 | 2116 | 1679 | 1417 | 1237 | 1124 | 982 | 892 | 828 | 780 | | | | | | | | | | | |
| | 50000 | 2422 | 1923 | 1679 | 1333 | 1124 | 982 | 892 | 780 | 708 | 657 | 619 | | | | | | | | | | | |
| | 70000 | 2165 | 1719 | 1501 | 1192 | 1005 | 878 | 798 | 697 | 633 | 588 | 553 | | | | | | | | | | | |
| | 100000 | 1923 | 1526 | 1333 | 1058 | 892 | 780 | 708 | 619 | 562 | 522 | 491 | | | | | | | | | | | |
| 2-3/4" 70 mm | 10000 | 4504 | 3575 | 3123 | 2479 | 2091 | 1826 | 1659 | 1450 | 1317 | 1223 | 1150 | | | | | | | | | | | |
| | 25000 | 3319 | 2634 | 2301 | 1826 | 1540 | 1346 | 1223 | 1068 | 970 | 901 | 848 | | | | | | | | | | | |
| | 50000 | 2634 | 2091 | 1826 | 1450 | 1223 | 1068 | 970 | 848 | 770 | 715 | 673 | | | | | | | | | | | |
| | 70000 | 2355 | 1869 | 1633 | 1296 | 1093 | 955 | 867 | 758 | 688 | 639 | 601 | | | | | | | | | | | |
| | 100000 | 2091 | 1659 | 1450 | 1150 | 970 | 848 | 770 | 673 | 611 | 567 | 534 | | | | | | | | | | | |
| 2-15/16" 3" 75 mm | 10000 | 4881 | 3874 | 3384 | 2686 | 2265 | 1979 | 1798 | 1571 | 1427 | 1325 | | | | | | | | | | | | |
| | 25000 | 3596 | 2854 | 2493 | 1979 | 1669 | 1458 | 1325 | 1157 | 1051 | 976 | | | | | | | | | | | | |
| | 50000 | 2854 | 2265 | 1979 | 1571 | 1325 | 1157 | 1051 | 919 | 835 | 775 | | | | | | | | | | | | |
| | 70000 | 2551 | 2025 | 1769 | 1404 | 1184 | 1035 | 940 | 821 | 746 | 693 | | | | | | | | | | | | |
| | 100000 | 2265 | 1798 | 1571 | 1247 | 1051 | 919 | 835 | 729 | | | | | | | | | | | | | | |



Table 5.7 (3) Radial Load/Speed Chart

Heavy Duty

Unit : lbf

Table with columns: SHAFT SIZE, L10, RPM (min-1) and rows for various shaft sizes (1", 1-1/2", 2", 2-1/2", 3", 3-1/2", 4", 5") and speeds (50, 100, 150, 200, 250, 300, 350, 400, 450, 500, 550, 6000, 6500).

Table with columns: SHAFT SIZE, L10, RPM (min-1) and rows for various shaft sizes (80 mm, 85 mm, 90 mm, 95 mm, 100 mm, 105 mm, 110 mm, 120 mm, 130 mm, 140 mm) and speeds (50, 100, 150, 200, 250, 300, 350, 400, 450, 500, 550, 6000, 6500).

1. Shaded Area A non-contact seal is used. If the bearing with set screws is exposed to a heavy load (Pr / Cr > 0.12), vibration, or heavy impact, use a tighter shaft tolerance than normal.

Ball Bearing Life Calculations

The relationship between the basic rating life, the basic dynamic load rating, and the dynamic equivalent load of the ball bearing is indicated in Formula 1. If the ball bearing unit is being used at a fixed rotating speed, the life is indicated as time. This is shown in Formula 2.

1. L10 = (Cr / Pr)^3
L10 : Basic Rating Life 10^6 rotations
L10h : Rated Life (hr)
Cr : Basic Dynamic Load Rating (kN)
Pr : Dynamic equivalent Load (kN)
n : Speed (min^-1)
2. L10h = (10^6 L10) / (60n) = (10^6 / 60n) * (Cr / Pr)^3

Table with columns: Basic Loads (lbf), SIZE, Dynamic Load (Cr), Static Load (Cor) for UC201 to UC218.

Table with columns: Basic Loads (lbf), SIZE, Dynamic Load (Cr), Static Load (Cor) for UC305 to UC328.

Table with columns: Basic Loads (lbf), SIZE, Dynamic Load (Cr), Static Load (Cor) for UCX05 to UCX20.